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Polar representation of material elasticity applied to wood and engineered wood products

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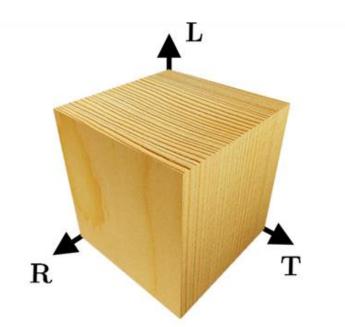
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Wood



 3 anatomical growth directions (L, R, T)
 Anisotropic / Orthotropic: Assuming orthogonal system in clearwood

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Background / History

- W. Voigt (1910): "Lehrbuch der Kristallphysik"
- J. Bodig, B. A. Jayne (1982): "Mechanics of Wood and Wood Composites"
- G. Schickhofer (1994): "Starrer und nachgiebiger Verbund bei geschichteten, flächenhaften Holzstrukturen"
- M. Grimsel (1999): "Mechanisches Verhalten von Holz : Struktur- und Parameteridentifikation eines anisotropen Werkstoffes"
- D. Keunecke, S. Hering, P. Niemz (2008): "Three-dimensional elastic behavior of common yew and norway spruce"

(List of selected works)

What is missing?

- Comprehensive derivations, "hard to understand" for inexperienced readers
- No substantial use of the potential of polar representation for parametric analysis

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Method

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Generalized Hooke's law

- 2nd order stress tensor: $\sigma \equiv \sigma_{ij} \mathbf{e}_i \otimes \mathbf{e}_j$ 2nd order strain tensor: $\varepsilon \equiv \varepsilon_{kl} \mathbf{e}_k \otimes \mathbf{e}_l$ \rightarrow $\sigma = \mathbf{C}\varepsilon$

• 4th order stiffness tensor:

$$\mathbf{C} \equiv C_{ijkl} \mathbf{e}_i \otimes \mathbf{e}_j \otimes \mathbf{e}_k \otimes \mathbf{e}_l$$

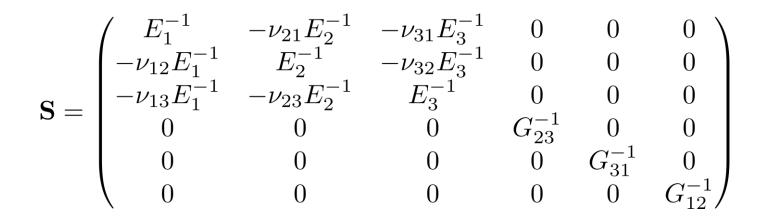
Compliance tensor: $S \equiv C^{-1}$

In matrix-notation & orthotropic material

$$\mathbf{S} = \begin{pmatrix} S_{11} & S_{12} & S_{13} & 0 & 0 & 0 \\ S_{21} & S_{22} & S_{23} & 0 & 0 & 0 \\ S_{31} & S_{32} & S_{33} & 0 & 0 & 0 \\ 0 & 0 & 0 & S_{44} & 0 & 0 \\ 0 & 0 & 0 & 0 & S_{55} & 0 \\ 0 & 0 & 0 & 0 & 0 & S_{66} \end{pmatrix}$$

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Orthotropic compliance matrix



> 9 (independent) engineering constants for orthotropic materials

Determined by experimental tests

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3D rotation of a compliance matrix?

2 Options:

- Rotation as a 4th-order tensor, using 4thorder tensor rotation rules and subscript notation, followed by a backwards transformation of the rotated tensor into Voigt matrix notation
- Direct rotation of the compliance matrix in Voigt notation (Bond 1943*)

Easier

*W. L. Bond, The mathematics of the physical properties of crystals, Bell System Technical Journal 22 (1) (1943) 1–72. doi:10.1002/j.1538-7305.1943.tb01304.x

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Simplified method

Method by Bond 1943* :

$$\mathbf{R} = \begin{pmatrix} R_{11} & R_{12} & R_{13} \\ R_{21} & R_{22} & R_{23} \\ R_{31} & R_{32} & R_{33} \end{pmatrix}$$

Any arbitrary rotation matrix

Rotation of a 6x6 Voigtnotation matrix becomes:

$$\mathbf{R}^{V} = \begin{pmatrix} R_{11}^{2} & R_{12}^{2} & R_{13}^{2} & R_{12}R_{13} & R_{13}R_{11} & R_{11}R_{12} \\ R_{21}^{2} & R_{22}^{2} & R_{23}^{2} & R_{22}R_{23} & R_{23}R_{21} & R_{21}R_{22} \\ R_{31}^{2} & R_{32}^{2} & R_{33}^{2} & R_{32}R_{33} & R_{33}R_{31} & R_{31}R_{32} \\ 2R_{21}R_{31} & 2R_{22}R_{32} & 2R_{23}R_{33} & R_{22}R_{33} + R_{23}R_{32} & R_{21}R_{33} + R_{23}R_{31} & R_{22}R_{31} + R_{21}R_{32} \\ 2R_{31}R_{11} & 2R_{32}R_{12} & 2R_{33}R_{13} & R_{12}R_{33} + R_{13}R_{32} & R_{13}R_{31} + R_{11}R_{33} & R_{11}R_{32} + R_{12}R_{31} \\ 2R_{11}R_{21} & 2R_{12}R_{22} & 2R_{13}R_{23} & R_{12}R_{23} + R_{13}R_{22} & R_{13}R_{21} + R_{11}R_{23} & R_{11}R_{22} + R_{12}R_{12} \end{pmatrix}$$

*W. L. Bond, The mathematics of the physical properties of crystals, Bell System Technical Journal 22 (1) (1943) 1–72. doi:10.1002/j.1538-7305.1943.tb01304.x

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Polar representation

 $r \in [0,\infty)$ $\varphi \in [0, 2\pi)$ \mathbf{e}_{z} $\theta \in [0,\pi)$ \mathbf{e}_{arphi} H \mathbf{e}_{ϵ} \mathbf{e}_y e

Material compliance is a vector **r** that can be rotated around in space depending on direction of acting load parametrized by φ and θ

$$\mathbf{r} = (\mathbf{e}_x, \mathbf{e}_y, \mathbf{e}_z) \begin{pmatrix} x \\ y \\ z \end{pmatrix} \equiv x \mathbf{e}_x + y \mathbf{e}_y + z \mathbf{e}_z$$
$$\mathbf{r} = (\mathbf{e}_x, \mathbf{e}_y, \mathbf{e}_z) \begin{pmatrix} r \sin \theta \cos \phi \\ r \sin \theta \sin \phi \\ r \cos \theta \end{pmatrix}$$
$$\mathbf{r} = (\mathbf{e}_x, \mathbf{e}_y, \mathbf{e}_z) \mathbf{R}_{\varphi \theta} \begin{pmatrix} 0 \\ 0 \\ r \end{pmatrix} \equiv (\mathbf{e}_\theta, \mathbf{e}_\varphi, \mathbf{e}_r) \begin{pmatrix} 0 \\ 0 \\ r \end{pmatrix}$$

Rotation matrix

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Rotation of compliance matrix

In conjunction with polar representation:

$$\mathbf{R} = \begin{pmatrix} \cos\varphi\sin\theta & \sin\varphi\sin\theta & \cos\theta \\ \cos\varphi\cos\theta & \cos\theta\sin\varphi & -\sin\theta \\ -\sin\varphi & \cos\varphi & 0 \end{pmatrix} \longrightarrow \begin{array}{c} \text{Build } \mathbf{R}^V \\ \text{using Bond rule} \end{array}$$

The 3D rotation of the compliance matrix reads:

$$\mathbf{S}' = \mathbf{R}^V \mathbf{S}(\mathbf{R}^V)^\intercal$$

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Rotation of uniaxial load

$$\mathbf{S}' = \mathbf{R}^V \mathbf{S}(\mathbf{R}^V)^\intercal$$

Yields

Material compliance is a vector **r** that can be rotated around in space depending on direction of acting load parametrized by φ and θ

If uniaxial load (tensile or compression), then element S_{11} of **S** is rotated

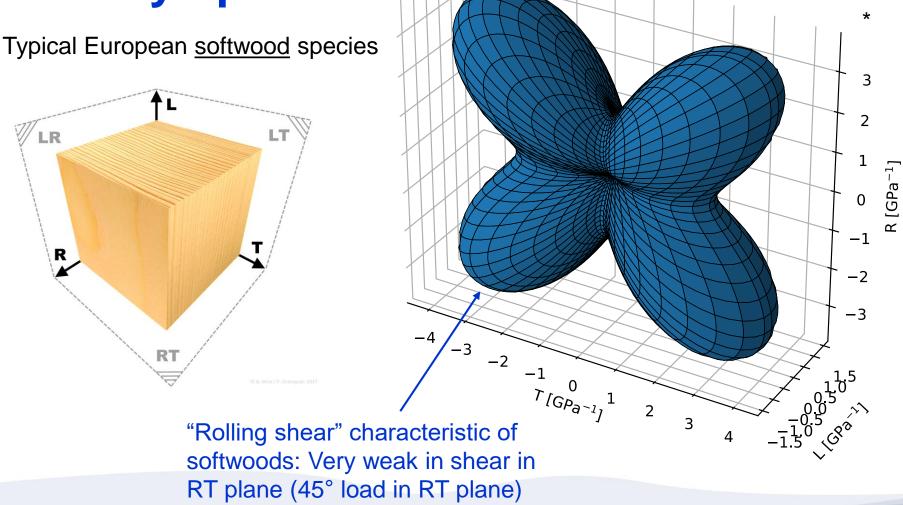
$$S'_{11} = S_{11} \cos^4 \varphi \sin^4 \theta + S_{22} \sin^4 \varphi \sin^4 \theta + S_{33} \cos^4 \theta$$
$$+ 2S_{12} \sin^2 \varphi \cos^4 \varphi \sin^4 \theta + 2S_{13} \cos^2 \varphi \sin^2 \theta \cos^2 \theta$$
$$+ 2S_{23} \sin^2 \varphi \sin^2 \theta \cos^2 \theta + S_{44} \sin^2 \varphi \sin^2 \theta \cos^2 \theta$$
$$+ S_{55} \cos^2 \varphi \sin^2 \theta \cos^2 \theta + S_{66} \cos^2 \varphi \sin^2 \varphi \sin^4 \theta$$



Educational example "Clearwood"

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Norway spruce



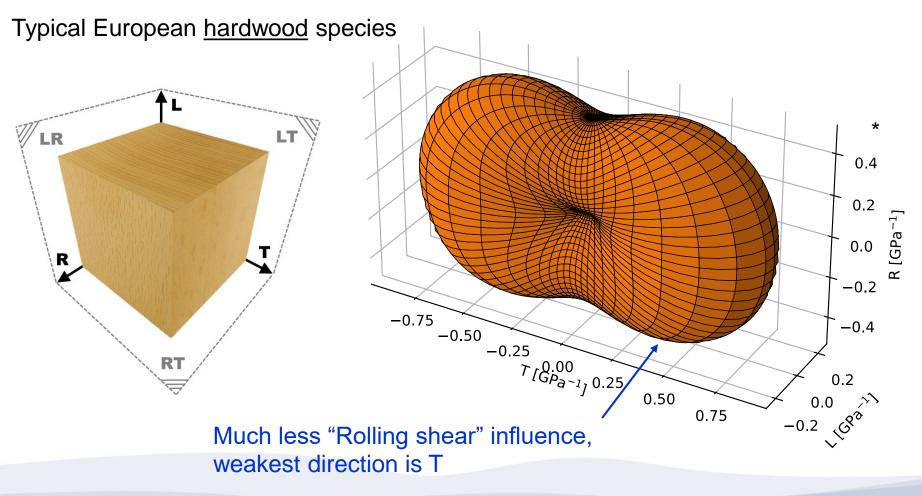
*Using literature data parameters for clearwood at 12% MC

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Interpretation 3 2 1 R [GPa⁻¹] 0 -1 -2 -3 -4 -3 -2 -1 0 $r_{[GP_{a}-1]}$ 1 2 3 4 Load in wood fiber direction, least compliant

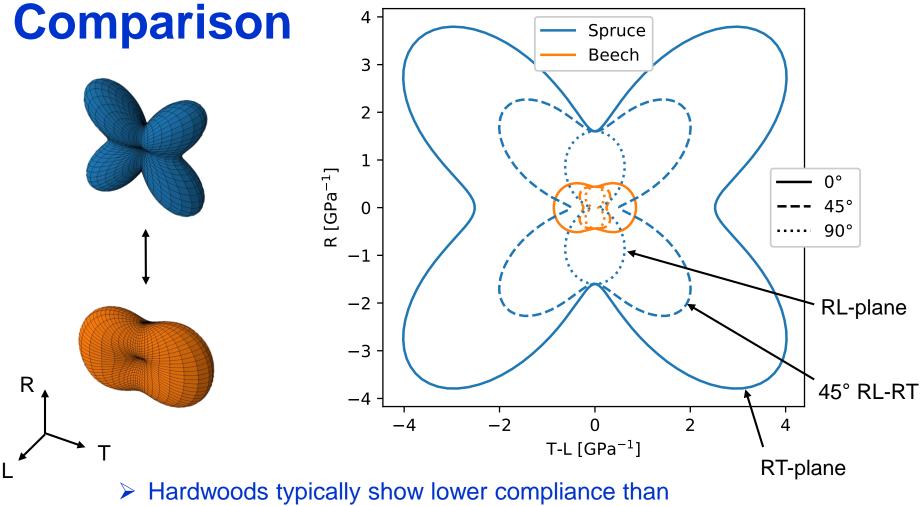
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European beech



*Using literature data parameters for clearwood at 12% MC

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- softwoods due to higher density
- Wood structure differences strongly affect rolling shear

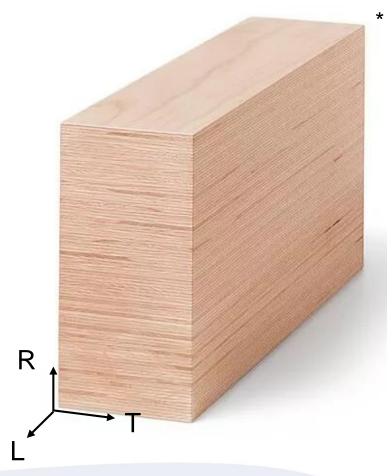
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Educational example Beech laminated veneer lumber (LVL)

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Beech LVL



High-performance engineered wood product, increasing use

- Main Applications: Columns, beams, reinforcements
- Wood anisotropy is somewhat retained due to veneer lamination, but very high "homogenization / lamination effect" (strength & stiffness)

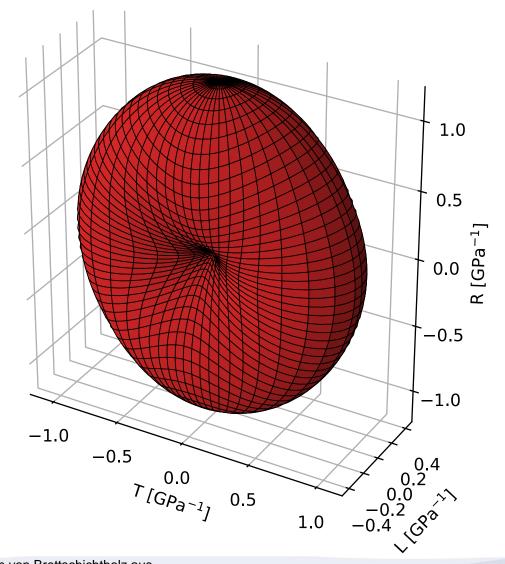
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Beech LVL

BauBuche (S) GL75 (ETA-14/0354)

WMC = 5.5%, n = 6-12*

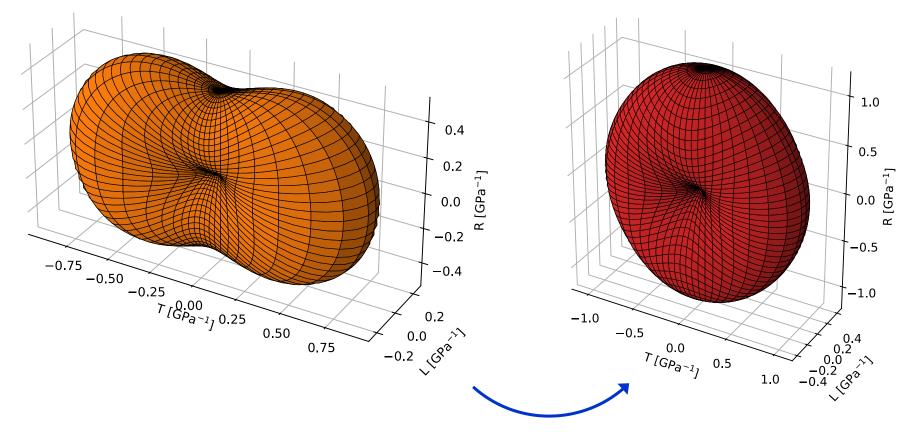
 $E_L = 17'259 \text{ MPa}$ $E_R = 840 \text{ MPa}$ $E_T = 966 \text{ MPa}$ $G_{LR} = 909 \text{ MPa}$ $G_{LT} = 1'006 \text{ MPa}$ $G_{RT} = 365^1 \text{ MPa}$ $\nu_{LR} = 0.305 \text{ MPa}$ $\nu_{LT} = 0.500 \text{ MPa}$ $\nu_{RT} = 0.208 \text{ MPa}$



*U. Kuhlmann, J. Töpler, Experimentelle und numerische Untersuchungen von Brettschichtholz aus Buchen-Furnierschichtholz (BauBuche), Doktorandenkolloquium Holzbau "Forschung und Praxis" 2022 ¹: Assumption from beech clearwood

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Beech clearwood vs. LVL



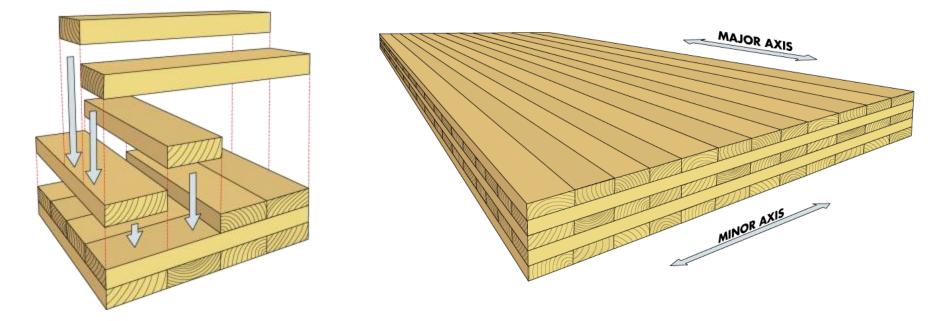
Homogenization in transverse directions, beech LVL more compliant
 Beech LVL less compliant in L direction ("homogenization/lamination effect")



Practical example Cross-laminated timber (CLT)

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Cross-laminated timber (CLT)



One of the most relevant engineered wood products, increasing use worldwide

- Main Applications: Slabs and walls
- Wood anisotropy is somewhat retained due to major vs minor axis

CLT calculation: Bending

Д.

"Gamma method" according to Eurocode 5 (EN 1995-1-1) for 5-layer CLT:

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$$\gamma_{1} = \frac{1}{1 + \frac{\pi^{2} E_{1} d_{1} d_{12}}{l_{ref}^{2} G_{R,12}}}} \qquad a_{1} = \left(\frac{a_{1}}{2} + d_{12} + \frac{a_{2}}{2}\right) - a_{2}$$

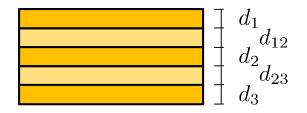
$$\gamma_{2} = 1.0 \qquad a_{2} = \frac{\gamma_{1} E_{1} d_{1} \left(\frac{d_{1}}{2} + d_{12} + \frac{d_{2}}{2}\right) - \gamma_{3} E_{3} d_{3} \left(\frac{d_{2}}{2} + d_{23} + \frac{d_{3}}{2}\right)}{\sum_{i=1}^{3} \gamma_{i} E_{i} d_{i}}$$

d.

$$\gamma_3 = \frac{1}{1 + \frac{\pi^2 E_3 d_3 d_{23}}{l_{ref}^2 G_{R,23}}} \qquad a_3 = \left(\frac{a_2}{2} + d_{23} + \frac{a_3}{2}\right) + \frac{1}{2} \left(\frac{a_3}{2} + \frac{a_3}{2}\right) + \frac{1}{2$$

Effective bending E-modulus

$$EI_{ef} = \sum_{i=1}^{3} E_i \frac{d_i^3}{12} + \sum_{i=1}^{3} \gamma_i d_i a_i^2 \longrightarrow E_{ef} = \frac{EI_{ef}}{I_0} = \frac{EI_{ef}}{\frac{(d_1 + d_{12} + d_2 + d_{23} + d_3)^3}{12}}$$



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Example parameters for CLT made of softwood C24 graded boards:

• E-Modulus in fiber direction:

$$E_{long} = 11'000 \text{ N/mm}^2$$

• Rolling shear modulus:

$$G_{R,ii} = 50 \text{ N/mm}^2$$

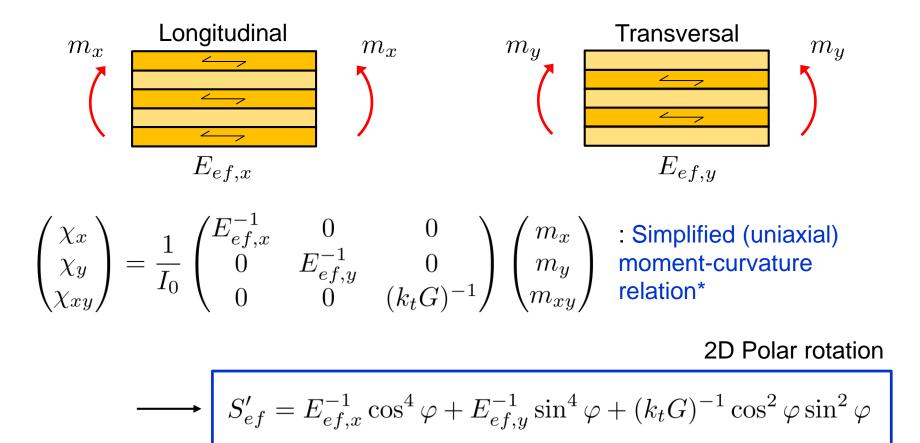
$$= EI_{ef}$$

Effective bending E-modulus

$$E_{ef} = \frac{EI_{ef}}{I_0} = \frac{EI_{ef}}{\frac{(d_1 + d_{12} + d_2 + d_{23} + d_3)^3}{12}}$$

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CLT: Rotation of uniaxial bending moment



* No coupling terms with axial forces (no biaxial moments), no lateral strain (v=0)

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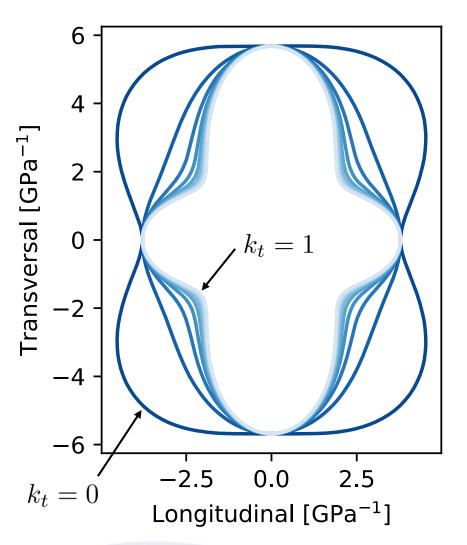
CLT: Edge-gluing

Example analysis: Effect of edgegluing of layers

 Variation of torsional moment factor:

Typical assumptions in practice:

- Edge-glued CLT: $k_t = 0.65$
- Non-edge-glued: $k_t = 0$



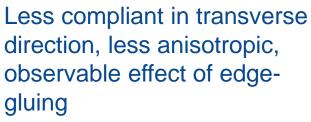
Strong axis" changes whether edge-glued or not!

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Example analysis: Effect of layer sizes / configuration in 5-layer CLT:

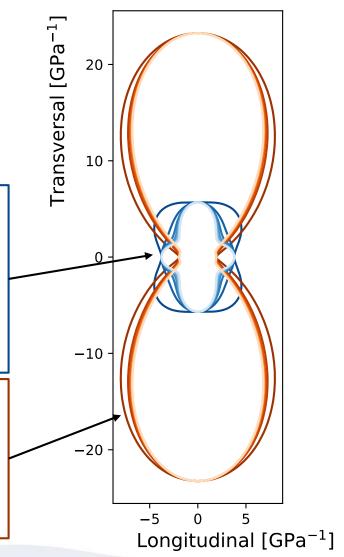




 $[40_0, 20_{90}, 40_0, 20_{90}, 40_0] \text{ mm}$



Less compliant in longitudinal direction, more anisotropic, no effect of edge-gluing



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Conclusions

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Conclusions

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- Polar representation is a powerful tool for visualization of anisotropic material behavior (e.g. wood, engineered wood products)
- Polar representation is not new, but its potential still remains somewhat unexploited
 - Science education for students: Building intuition and comprehension for material behavior
 - Materials selection and design for engineers: Compare and optimize design solutions, assess different effects



Thanks for your attention!

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